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54 Focal length switchover camera.

57 A camera comprising a photographic optical system adjustable to a first focal length adjustment and to a second focal length adjustment, control means allowing a switch of said photographic optical system from said first focal length adjustment to said second focal length adjustment, and allowing focus adjustments of said photographic optical system in its first and second focal length adjustments, and leading means operatively connected with said control means, further comprises a first lead portion formed on said leading means so as to allow in said first focal length adjustment of said photographic optical system a first focus adjustment from a first shortest distance setting to a first longest distance setting, a second lead portion formed on said lead-

ing means and disposed successively to said first lead portion in a direction of operation of said leading means, so as to allow a switch of said photographic optical system from said first focal length adjustment to said second focal length adjustment, a third lead portion formed on said leading means and disposed successively to said second lead portion in a direction of operation of said leading means, so as to allow in said second focal length adjustment of said photographic optical system a second focus adjustment from a second shortest distance setting to a second longest distance setting, and follower means driven by said first to third lead portions and connected with said photographic optical system so as to drive the same.

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The present invention relates to a focal length switchover camera capable of e.g. a standard photography and a tele photography or the standard photography and a wideangle photography, and more particularly to a camera of the above-mentioned type having control means capable of effecting a switchover of the focal length of and a focus adjustment of a photographic optical system in a serial and continuous manner.

There is known a focal length switchover camera of the above-mentioned type having control means constituted by a screw member extending in a direction of optical axis and a mount plate of the photographic optical system engaging the screw member. In the case of this conventional camera, the focal length switchover operation and the focus adjustment operation in the respective focal length conditions are carried out by drivably rotating the screw member by means of a motor or the like thereby moving the photographic optical system mounted on the mount plate in the direction of the optical axis (e.g. a Japanese patent application published under SHOWA 60-122931).

However, this camera has problems to be described hereinafter.

The amount of movement of the photographic optical system for switching over the focal length thereof is generally larger than that of the same system for effecting a focus adjustment in the respective focal length conditions. Also, this movement amount of the photographic optical system for effecting a focus adjustment in the respective focal length conditions increases with an increase in the focal length.

Therefore, in the conventional control means, the screw member is employed for moving and fixedly positioning the photographic optical system. This screw member has a thread pitch uniform over its whole length and thus the movement amount of the photographic optical system relative to the rotational amount of the screw member is always the same. Therefore, unless the motor is of the variable speed type thereby complicating the control of the same, if the thread pitch of the screw member is formed small for effecting with a high precision the focus adjustment operation in the shorter focal length condition in which the movement amount of the photographic optical system is small, the focus adjustment operation and the focal length switchover operation in the longer focal length condition takes a long time. On the other hand, if the thread pitch of the screw member is formed large for effecting more speedily the focal length switchover operation and the focus adjustment operation in the longer focal length condition in which the movement amount of the photographic optical system is large, it becomes difficult to effect with a high precision the focus adjustment opera-

tion in the shorter focal length condition.

Further, generally in the case of the camera of the above type, it is necessary to detect a delivery position of the photographic optical system in effecting a focus adjustment operation and thus there is provided a detecting device for detecting this position of the photographic optical system. A conventional position detecting device of this kind includes a position detecting member attached to the photographic optical system and a sensor for detecting a passage or a contact of the member. In this case, it is necessary to reserve space specially for disposing the member to be attached to the photographic optical system and for the sensor, whereby it becomes difficult to form the camera compact.

There is also known a focal length switchover mechanism for switching over a focal length by sliding along an optical axis a fore-element of the photographic optical system between a position projected from a camera body and a position retracted from the same.

This focal length switchover mechanism includes a screw member extending in the direction of the optical axis and a mount plate for the optical system. In operation, the focal length switchover operation and the focus adjustment operation in the respective focal length conditions are carried out by drivably rotating the screw member by means of a motor or the like thereby moving the optical system mounted on the plate in the direction of the optical axis.

However, in the case of a camera having such focal length switchover mechanism, the movement amount of the fore-element for switching over the focal length is relatively large. Thus, a member, e.g. a movable lens barrel, for supporting the fore-element at its projected position projects considerably from the camera body. Accordingly, this projecting portion of the camera is very likely to be subjected to external shocks, and especially when the camera is placed with the fore-element being positioned downwardly, the total weight of the camera body is applied to this projecting portion. Moreover, since the screw member of the focal length switchover mechanism and the mount plate for the fore-element are constantly coupled with each other, if, as described above, any shocks or external forces are applied to the portion considerably projected from the camera body, all the shocks and forces are directly transmitted to the focal length switchover mechanism thereby often damaging the same.

Taking the above state of the art into consideration, it is the primary object of the present invention to provide an improved focal length switchover camera capable of effecting with a high precision a focus adjustment operation in a short focal length

According to a still further development of the invention, there is provided a camera capable of switching over a focal length as at least a front portion of a photographic optical system shifts along a direction of an optical axis thereof between a position projected from a camera body and a position receded from the camera body. This camera comprises an urging member for urging the front portion towards the position projected from the camera body and a control mechanism capable of shifting the front portion towards the position receded from the camera body against an urging force of the urging member and capable of permitting the front portion to further shift away from the mechanism towards the direction exceeding the shift amount.

With the above construction, when the fore-element is at its projected position, even if the member holding this fore-element is subjected to shocks or to the total weight of the camera body when the camera is placed with this fore-element side being positioned downwardly, since the controlled member of the control mechanism constituting the focal length switchover mechanism is moved towards the receded position relative to the controlling member against the urging force of the urging member, it becomes possible to prevent the focal distance switchover mechanism from being subjected to any extreme external forces.

With disappearance of the force, the controlled member of the control mechanism returns to its original position by the urging force of the urging member and the fore-element of the photographic optical system also returns to its original position thereby the focal length returns to its original state.

Accordingly, it becomes possible to prevent any damages to the focal length switchover mechanism without interfering with a smooth photography operation.

The present invention is hereinafter more specifically described with reference to exemplary embodiments depicted in the drawings, in which;

Fig. 1 is an overall plane view,

Fig. 2A is an exploded perspective view of a photographic optical system,

Fig. 2B is an exploded perspective view of a finder optical system,

Figs. 3A and 3B are back view of a shutter block with Fig. 3A showing a standard photography condition and Fig. 3B showing a tele photography condition,

Fig. 4 is a back view showing in an enlarged scale a main portion of the shutter block in the standard photography condition,

Fig. 5 is an exploded perspective view showing the shutter block, a movable lens barrel and a portion of projecting and receding means for an auxiliary lens unit,

Fig. 6 is an exploded perspective view showing the portion of the projecting and receding means for the auxiliary lens unit,

Fig. 7 is a side view showing in an enlarged scale the portion of the projecting and receding means for the auxiliary lens unit,

Fig. 8 is a perspective view of a cam face of a fixing mount plate,

Figs. 9A through 9C are schematic plane views illustrating relationship among the shutter block, the movable lens barrel and the projecting and receding means for the auxiliary lens unit, with Fig. 9A showing the standard photography condition, Fig. 9B showing focal length switchover operation and Fig. 9C showing the tele photography condition, respectively,

Fig. 10 is a perspective view showing the photographic optical system in the standard photography condition,

Fig. 11 is a perspective view showing a delivery cam and a delivery lever,

Fig. 12 is a sectional view showing the delivery cam and the delivery lever,

Fig. 13 is a plane view showing the delivery cam,

Fig. 14 is a developed view a cam face of the delivery cam for a focus adjustment in a short focal length condition,

Fig. 15 is a plane view showing adjustment means of the delivery lever for a support lever,

Fig. 16 is a plane view showing position detecting brushes of the delivery cam,

Fig. 17 is a plane view of a position detecting pattern of a gear mount plate,

Fig. 18 is a bottom view of the gear mount plate,

Fig. 19 is a diagram showing a peripheral circuit of a control device,

Figs. 20A through 20E are plane views showing a drive mechanism for the photographic optical system, with Fig. 20A showing an initial condition in which the photographic optical system is focused at its infinity position in the standard photography condition, Fig. 20B showing a condition in which the photography optical system is focused at its closest focusing position in the standard photography condition, Fig. 20C showing the drive mechanism in the course of a focal length switchover operation, Fig. 20D showing an initial condition in which the photographic optical system is focused at its infinity position in the tele photography condition and Fig. 20E showing a condition in which the photography optical system is focused at its closest focusing position in the tele photography condition, respectively,

Fig. 21 is a flow chart illustrating a focus adjustment operation,

lens unit 4a used both in the standard photography condition and the tele photography condition, the main lens frame 4b holding this main lens unit 4a as a for-element of the photographic optical system PS and a shutter not shown. The shutter block 4 is supported by a pair of shafts 5X and 5Y secured to a fixing mount plate 5 to be freely slidable in the direction of an optical axis L and at the same time the block 4 is forwardly urged by means of a spring 6.

To this shutter block 4, there is attached an auxiliary lens unit 7 supported by a support lever 7A freely rotatable about an axis X1 parallel to the optical axis L to be inserted into a photographic optical path only in the tele photography condition. Further, to this shutter block 4, there is fixedly attached a movable lens barrel 8 covering the auxiliary lens unit 7 and inwardly engaged with a lower half portion 5A which is a first attaching frame shaped as a cylinder having an angular or circular cross section for the fixing mount plate 5 formed integrally therewith, whereby both the movable lens barrel 8 and the shutter block 4 are slidable in the direction of the optical axis L and forwardly urged by the spring 6.

As shown in Figs. 3A, 3B and also in Fig. 4, the support lever 7A for the auxiliary lens unit 7 is urged by means of a spring 9 to move the auxiliary lens unit 7 into the photographic optical path. In an outer peripheral face of an axis portion of the support lever 7A, there is formed a gear portion 7a. There is also provided a fan-shaped control member 11 pivotable about an axis X2 in unison with a gear member 10 having a gear portion 10a meshing the gear portion 7a.

As shown in Figs. 5 and 6, this control member 11 includes an axis portion 11a having an elliptical cross section. This axis portion 11a is inserted through a circular hole 10b of the gear member 10 to be rotatable relative to each other and the portion 11a is further engaged into an elliptical hole 12a of an intermediate member 12 to be unrotatable relative to each other. As shown in Fig. 5 and 7, an arm portion 12b of this intermediate member 12 and an arm portion 10c of the gear member 10 are superposed on each other and there is provided a coil spring 13 having bent ends for binding the both arm portions.

Referring back to Fig. 4, as the control member 11 is rotated clockwise, the intermediate member 12 and the gear member 10 coupled by means of a spring with the member 12 are rotated in the same direction. On the other hand, through meshing between the gear portion 10a of the gear member 10 and the gear portion 7a of the support lever 7A, the support lever 7A is rotated counterclockwise, whereby the auxiliary lens unit 7 is recessed from the photographic optical path.

In this condition, the control member 11 as a controlling member is coupled by means of spring with the gear member 10 as a controlled member to be rotatable relative to each other so that the control member 11 may be freely moved in a direction that the member 11 retracts the auxiliary lens unit 7 even after the holder 7B of the auxiliary lens unit 7 comes into contact with an upper frame 8a of the movable lens barrel 8. With the above construction, it is possible to minimize the space for causing the auxiliary lens unit 7 to recede from the photographic optical path and at the same time to reserve a large operative range for the control member 11. As the result, the auxiliary lens unit 7 may be reliably recessed from the optical path in spite of possible manufacturing or mounting tolerances of the components constituting the receding mechanism for the auxiliary lens unit 7.

As shown in Figs. 5 and 10, this control member 11 projects from an upper face of the movable lens barrel 8 to the outside. And, as shown in Figs. 8 and 10, inside the fixing mount plate 5, there is integrally formed a cam face 5a to come into contact with a controlling projection 11b of the control member 11 with a movement of the shutter block 4 in the direction of the optical axis L thereby causing the auxiliary lens unit 7 to recede from the photographic optical path.

In the standard photography condition, as shown in Figs. 3A, 9A and 10, the controlling projection 11b of the control member 11 is in contact with a flat portion 5x of the cam face 5a of the fixing mount plate 5. In this condition, the control member 11 is rotated to and held at its clockwise extreme end position, and the auxiliary lens unit 7 is recessed from the photographic optical path against the urging force of the spring 9.

The flat portion 5x of the cam face 5a is formed in parallel with the optical axis L and has a certain length in this direction. That is to say, when the shutter block 4 and the movable lens barrel 8 are moved forwardly or backwardly in the direction of the optical axis L in order to effect a focus adjustment as will be more particularly described later, the auxiliary lens unit 7 is fixedly maintained at its recessed position from the photographic optical path.

The cam face 5a has a curved portion 5y curved so as to depart from the optical axis L following the flat portion 5x. From the above-described standard photography position, if the shutter block 4 and the movable lens barrel 8 are together with moved forwardly along the optical axis L, the controlling projection 11b of the control member 11 rotationally urged counterclockwise in connection with the support lever 7A rotationally urged clockwise by means of the spring 9 in Figs. 3A, 3B and 4, comes out of its contact with the flat

As shown in Fig. 11, since the pivotal movement of the delivery lever 18 towards its recessed position is permitted as the pair of pins 18b and 18c of the delivery lever 18 corresponding respectively to the pair of cam faces 14b and 14c of the delivery cam 14 come into one-side contact with the same, even if the shutter block 4 for holding the main lens unit 4a in the tele photography condition is subjected to an external force and e.g. the camera is placed with the shutter block 4 being positioned downwardly, the lever 18 is oscillated to leave the delivery cam 14. Therefore, it is possible to prevent the control mechanism from being subjected to an extreme external force, thereby protecting the mechanism from the damages.

As will be more particularly described later in the description of operations, with a normal or reverse rotation of the motor M, the delivery cam 14 is rotated in a normal or reverse direction and in connection with this rotation, the delivery lever 18 is shifted to fixedly position the shutter block 4 through the movable lens barrel 8, whereby the main lens unit 4a is moved along the direction of the optical axis L and the auxiliary lens unit 7 is projected into or receded from the photographic optical path.

The drive of the motor M also effects the focal length switchover between the standard photography condition and the tele photography condition, and further, in the two respective photography conditions, the drive of the motor M moves the main lens unit 4a thereby effecting a focus adjustment operation.

That is to say, the gear train 15, delivery cam 14, delivery lever 18, stop lever 16, magnet 17, motor M and so on together with constitute control means for effecting in succession the focal length switchover operation of the photographic optical system PS and the focus adjustment operation in the respective focal length condition.

In the above operations, by urging the shutter block 4 forwardly and fixedly positioning the same by the delivery lever 18 having the pair of pins 18b, 18c respectively coming into contact with the pair of cam faces 14b and 14c of the delivery cam 14, it is possible to provide the cam faces 14b and 14c with an inclination angle relative to the operative direction of the cam 14 suitable for a required moving speed of the shutter block 4 (e.g. the speed may be faster in a focus adjustment operation in the tele photography condition than in that in the standard photography condition, or the speed may be faster in a focal length switchover operation than in a focus adjustment operation) in a tangential direction of the rotational track of the delivery cam 14, i.e. the inclination angle of the cam faces 14b and 14c relative to the operative direction of the cam 14. As the result, the focal length

switchover operation in photographing may be carried out speedily and the focus adjustment operation may be carried out with a high precision.

The first cam face 14b of the delivery cam 14 and the second pin 18b of the control lever 18 coming into contact therewith effect the focus adjustment operation of the photographic optical system PS in the standard photography condition and a former half of the focal length switchover operation from the standard photography condition to the tele photography condition.

On the other hand, the second cam face 14c of the delivery cam 14 and the third pin 18c of the control lever 18 coming into contact therewith effect the latter half of the focal length switchover operation from the standard photography condition to the tele photography condition and also the focus adjustment operation in the tele photography condition.

More particularly, the first cam face 14b of the delivery cam 14 forms an end portion of a short focal length focus adjustment cam face 14X and a focal length switchover cam face 14Y; whereas, the second cam face 14c forms the other end of the focal length switchover cam face 14Y and a long focal length focus adjustment cam face 14Z.

These short focal length focus adjustment cam face 14X, focal length switchover cam face 14Y and long focal length focus adjustment cam face 14Z are disposed in this order in the rotational direction of the delivery cam 14, i.e. in the operative direction of this cam 14.

By dividing the cam face into the first cam face 14b and the second cam face 14c, it becomes possible to reduce operational errors of the delivery lever 18 acting as the cam follower.

That is to say, if the cam face is formed as one, an end of this cam face is formed adjacent a rotational axis X5 of the delivery cam 14. In this case, a peripheral length of the cam face relative to a predetermined rotational angle of the delivery cam 14 becomes relatively short. Then, the cam face formed adjacent the rotational axis X5 of the delivery cam 14 will have a large error relative to the same rotational angle error in comparison with the cam face formed adjacent the outer peripheral face of the delivery cam 14, and therefore, the delivery lever 18 will have a larger operational error.

In contrast to this, by dividing the cam face as described above and transferring this in the course of the operation, it becomes possible to form the cam face only adjacent the outer peripheral face of the delivery cam 14 thereby reducing the operational error as well.

Moreover, the first cam face 14b and the second cam face 14c constituting the divided peripheral faces of the rotational delivery cam 14

pin 18c of the delivery lever 18 and the second cam face 14c of the delivery cam 14.

On the other hand, the support lever 21 acting as a support member for rotatably supporting the delivery lever 18 by pivotably supporting the first pin 18a of the delivery lever 18, as shown in Figs. 2A and 15, outwardly engages a pin 23a implanted on the gear mount plate 23 to be rotatable about an axis X6.

The support lever 21 defines an elongate slot 21b opposed to the pivot portion 21a across the rotational axis X6 thereof. On the other hand, the gear mount plate 23 defines at a portion corresponding to the elongate slot 21b a hole 23j having a diameter larger than the short axis of the elongate slot 21b.

The support lever 21 is oscillatable about the axis X6. Referring to how to rotatable mount this lever 21, an adjustment tool having a first cylindrical shaft having an outer radius as large as the inner radius of the hole 23j of the gear mount plate 23 and a second cylindrical shaft eccentrically attached to a leading edge of the first cylindrical shaft and having an outer radius engageable with the elongate slot 21b of the support lever 21 is inserted into the hole 23j of the gear mount plate 23 from under with only the second cylindrical shaft extending through the elongate slot 21b, and then the tool is rotated.

In the same figures, a reference numeral 22 denotes a fixing screw which extends through another elongate slot 21c defined in the support lever 21, and a screw portion thereof engages the gear mount plate 23.

That is to say, the construction in which the support lever 21 is oscillatable about the axis X6, the position adjusting elongate slot 21b and the fixing screw 22 together with constitute adjustment means for varying the position of the support lever 21 acting as a support member for the control lever 18 as the cam follower.

As the support lever 21 is oscillated about the axis X6, the position of its pivot portion 21a also changes. For example, if the support lever 21 is oscillated counterclockwise from a position indicated by a solid line in Fig. 15, the pivot portion 21a moves downwardly as indicated by two-dot-and-dash line in Fig. 15.

In this case, since the fourth pin 18d is in engagement with the movable lens barrel 8 and the second pin 18b is fixedly positioned as coming into contact with the first cam face 14b of the delivery cam 14, the delivery lever 18 pivots counterclockwise about this second pin 18b. Therefore, as indicated by a two long and short dashes line in Fig. 15, the fourth pin 18d of the delivery lever 18 moves upwards in the same figure.

Thereby, the shutter block 4 and the movable lens barrel 8 urged forwardly in the direction of the optical axis L moves forwardly and then the main lens unit 4a changes its position.

In Fig. 15, the first pin 18b of the delivery cam 14 is positioned at a start end portions 14s of the first cam face 14b of the delivery cam 14. This position, through will be more particularly described later, corresponds to a condition in which the photographic optical system PS (in this case, the main lens unit 4a) is focused at its infinity position in the standard photography condition. In this condition, if there are no errors, the lens block 4 and the movable lens barrel 8 are positioned at the most rearward ends of their respective movable ranges.

And, as described hereinbefore, by changing the position of the main lens unit 4a through the oscillation of the support lever 21, regardless of errors due to manufacturing or mounting errors of the main lens unit 4a, it is possible to fine-adjust the position of the photographic optical system PS such that the system PS may be focused at the infinity position with the first pin 18b of the delivery lever 18 being placed at the start end portion 14s of the first cam face 14b of the delivery cam 14.

The position adjusting elongate slot 21b, one of the adjustment members constituting the adjustment means, is exposed to the outside of the control means. Therefore, the above-described fine adjustment operation of the photographic optical system PS may be readily carried out even after the control means is attached.

In Fig. 15, it is to be noted, there is shown the fine adjustment of the photographic optical system PS through the oscillation of the support lever 21 with the first pin 18b of the delivery lever 18 being in contact with the start end portion 14s of the first cam face 14b of the delivery cam 14. In place of this, it is also possible to carry out the fine adjustment with the second pin 18c of the delivery lever 18 being in contact with an end portion of the second cam face 14c of the delivery cam 14. This position corresponds to a condition in which the photographic optical system PS (in this case the main lens unit 4a and the auxiliary lens unit 7) is focused at its closest focusing position. In this condition, if there are no errors, the shutter block 4 and the movable lens barrel 8 are positioned at the forwardmost ends in their respective movable ranges.

As shown in Fig. 16, the lower face of the delivery cam 14 carries three position detecting brushes, a first brush 14A, a second brush 14B and a third brush 14C positioned differently each other in the peripheral direction thereof. On the other hand, as shown in Fig. 17, the upper face of the gear mount plate 23 acting as a fixing portion on

Also, only the lens initial condition detecting switch S3 is closed; whereas, the other switches S1, S2, S4 and S5 are opened (see step #0 in Fig. 21).

If the release switch S1 is closed with an operation of the release button 1 at step #1, the magnet 17 is energized to attract the iron piece 20 at step #2, and then at step #3 the motor M starts its counterclockwise rotation in Fig. 20A. Then, at step #4, through the gear train 15, the delivery cam 14 starts its clockwise rotation and the lens initial condition detecting switch S3 is opened.

With this rotation of the delivery cam 14, the second pin 18b of the delivery lever 18 moves along the first cam face 14b of the delivery cam 14 and the delivery lever 18 is rotated counterclockwise about the axis X4, whereby the shutter block 4 is moved forwardly in the direction of the optical axis L (upwardly in Fig. 20A) by the urging force of the spring 6.

Also with the rotation of the delivery cam 14, the lens position detecting switch S4 is repeatedly opened and closed at steps #5, #6, and then at step #7, the motor M is stopped when the photographic optical system PS has reached the focused position. At step #6, it is judged whether the photographic optical system PS has reached the focused position or not. This judgement is made by the control device 24 which counts the pulse signals generated by the repeated opening and closure of the lens position detecting switch S4 and then compares the counted value with a focus count value obtained independently.

After the motor M is stopped with its terminals being shorted out, the power supply to the magnet 17 is stopped at step #8. Thereby, as shown in Fig. 20B, the stop lever 16 is pivoted about the axis X3 by the urging force of the spring 19 and its claw 16a comes into engagement with the ratchet portion 14a of the delivery cam 14 to fixedly position this delivery cam 14.

At step #9, a timer (not shown) is started, and then a shutter (not shown) is opened and closed at step #10. At step #11, the process waits for a timing-up of the timer. This timer is set to time-up after a time lapse slightly longer than a maximum shutter speed. That is to say, the process waits for the completion of the shutter operation.

With this completion of shutter operation, the motor M starts its clockwise rotation in Fig. 20B at step #12. Then, through the gear train 15, the delivery cam 14 starts its counterclockwise rotation in the same figure. In accordance therewith, the delivery lever pivots clockwise and the shutter block 4 and the movable lens barrel 8 are moved backwardly in the direction of the optical axis L.

When the delivery cam 14 has its initial position, the lens initial condition detecting switch S3 is

closed at step #13. Thereafter, the motor is stopped with its terminals being shorted out at step #14 and the power supply to the magnet 17 is stopped at step #15.

Thereby, at step #16, the process retruns to the initial condition shown in Fig. 20A.

(2) focal length switchover operation

For switching over the photography conditions of the two focal distances, the focal distance switchover button 2 is operated. Various switchover operations will be respectively described next.

i) switchover from the standard photography condition to the tele photography condition

In the initial state of the standard photography condition shown in Fig. 20A (see step #100 in Fig. 22), if the lens switchover switch S2 is closed with a depression of the focal length switchover button 2 at step #101, the state of the lens area detecting switch S5 is checked at step #102. If the camera is in the standard photography condition, this lens area detecting switch S5 is opened. Thus, the process goes on to step #110.

At this step #110, the motor M effects its counterclockwise rotation in Fig. 20A, and then the magnet 17 is energized to attract the iron piece 20 at step #111. With this rotation of the motor M, the delivery cam 14 starts its clockwise rotation and the lens initial condition detecting switch S3 is opened at step #112.

With this clockwise rotation of the delivery cam 14, in the same manner as in the focus adjustment operation in the standard photography condition, the delivery lever 18 having its first cam face 14b in contact with the second pin 18b is rotated counterclockwise. Thereby, the shutter block 4 and the movable lens barrel 8 are moved forwardly in the direction of the optical axis L.

Since the first cam face 14b of the delivery cam 14 is not defined over the whole length of the same, as shown in Fig. 20C, if the delivery lever 18 is rotated and its second pin 18b passes the end portion 14t of the first cam face 14b, the first cam face 14b does not come into contact with the second pin 18b any longer. Therefore, by being pulled by the shutter block 4 urged forwardly in the direction of the optical axis L, the delivery lever 18 is permitted to be further rotated counterclockwise.

In this condition, the third pin 18c of the delivery lever 18 is in contact with the second cam face 14c of the delivery cam 14, and then, as the third pin 18c of the delivery lever 18 moves along the second cam face 14c with the clockwise rotation of the delivery cam 14, the rotation of the delivery lever 18 is controlled.

the delivery cam 14, thereby fixedly positioning the shutter block 4 urged forwardly in the direction of the optical axis L (the upper direction in the figure). Also, the second pin 18b of the delivery lever 18 is not in contact with the first cam face 14b of the delivery cam 14.

In the above condition, the lens initial condition detecting switch S3 and the lens area detecting switch S5 are closed and the other switches S1, S2, S4 are opened (see at step #0 in Fig. 21).

If the release switch S1 is closed with an operation of the shutter release button 1 at step #1, the magnet 17 is energized to attract the iron piece 20 at step #2, and then the motor M starts its rotation counterclockwise in Fig. 20D at step #3. Then, through the gear train 15, the delivery cam 14 starts its clockwise rotation and the lens initial condition detecting switch S3 is opened at step #4.

With this rotation of the delivery cam 14, the third pin 18c of the delivery lever 18 moves along the second cam face 14c of the delivery cam 14 and the delivery lever 18 is pivoted about the axis X4. Thereby, the shutter block 4 is moved forwardly in the direction of the optical axis L by the urging force of the spring 6.

Also with the rotation of the delivery cam 14, the lens position detecting switch S4 is repeatedly opened and closed at steps #5, #6, and then when the photographic optical system PS has reached in focus position, the motor is stopped with its terminals being shorted out at step #7, and thereafter the power supply to the magnet 17 is stopped at step #8.

Thereby, the stop lever 16 is pivoted about the axis X3 counterclockwise, as shown in Fig. 20E, by the urging force of the spring 19 and its claw 16a comes into engagement with the ratchet portion 14a of the delivery cam 14, whereby the delivery cam 14 is fixedly positioned.

Thereafter, the timer is started at step #9 and the shutter is opened and then closed at step #10. Then, the process waits for the completion of the shutter operation. With the completion of this shutter operation, the motor M starts its rotation clockwise in Fig. 20E at step #12.

Then, through the gear train 15, the delivery cam 14 starts its counterclockwise rotation. With this rotation, the delivery lever 18 is pivoted clockwise and the shutter block 4 and the movable lens barrel 8 are moved backwardly in the direction of the optical axis L.

When the delivery cam 14 has reached its initial position, the lens initial condition detecting switch S3 is closed at step #13. Thereafter, the motor M is stopped with its terminals being shorted out at step #14 and the power supply to the magnet 17 is stopped at step #15.

With this, the process returns to the initial condition shown in Fig. 20D at step #16.

During the above-described switchover operation between the standard photography condition and the tele photography condition in the photographic optical system PS, a switchover operation of magnification is also carried out in a view finder optical system FS. The construction for effecting this switchover operation will be described next.

As shown in Fig. 2B, Figs. 23A, 23B, and 23C, this view finder optical system FS includes an objective lens 25, a variable magnification lens 26, a view frame 27, an eyepiece lens 28, a pressing plate 29, a frame plate and so on. This view finder optical system FS is accommodated in an upper half portion 5B, which is a second mount frame shaped as a cylinder having an angular or circular cross section, of the integrally formed fixing mount plate 5.

As shown in Fig. 2B, this upper half portion 5B of the fixing mount plate 5 also accommodates a near infrared light emitting element 31 constituting a light emitting section for carrying out an automatic focus adjustment in the active system and a converging lens 32 and a light receiving element 33 together with acting as a light receiving section. Further, a frame member 34 including a light emitting window 34a, a light receiving window 34b and a view finder window 34c covering the fronts of the light emitting section, the light receiving section and the view finder optical system FS is securely mounted on the fixing mount plate 5.

As shown in Fig. 2B, 24A, 24B, and 24C, in the view finder optical system FS, an objective lens holder 35 holding the objective lens 25 includes a tubular portion 35A having an angular or circular cross section and formed along the direction of the optical axis L. This tubular portion 35A outwardly engages a guide shaft 36 secured to the fixing mount plate 5, such that the objective lens 25 is movable in the direction of the optical axis L.

Also, a variable magnification lens holder 37 holding the variable magnification lens 26 is attached to be pivotable about an axis X7, such that the variable magnification lens 26 is projectable and retracted relative to an optical path of the view finder. There is also provided a spring (not shown) for urging the variable magnification lens 26 into the view finder optical path via the variable magnification lens holder 37.

Fig. 23A and Fig. 24A show the standard photography condition. In this condition, as will be more particularly described later, the objective lens 25 is at a forwardmost position in its movable range, and the variable magnification lens 26 is inserted into the view finder optical path by the urging force of the spring.

same figure. In this condition, the photographic optical system PS is focused at its infinity position. If the motor M is driven from this initial condition, the bevel gear 40 is rotated clockwise to return to a position denoted by a reference mark "b" in the same figure.

In the course of the above movements, the focus adjustment operation in the standard photography condition is carried out. That is, at the position "b", the photographic optical system PS is focused at its closest focusing position.

Also, in the course of the above movements, the gear portion 40a of the bevel gear 40 is not in mesh with the bevel gear portion 39b of the switchover lever 3. Therefore, the magnification is not changed over in the view finder optical system FS and the system FS has a magnification suitable for the standard photography condition. In this condition, as shown in Fig. 27A, the switchover lever 39 is forwardly inclined.

((2)) focal length switchover operation

For effecting a focal length switchover from the standard photography condition to the tele photography condition, the bevel gear 40 is further rotated clockwise beyond the position "b" in Fig. 28A and its gear portion 40a reaches a position denoted by a reference mark "c" to nearly mesh with the bevel gear portion 39b of the switchover lever 39. In the above movements, the magnification in the view finder optical system FS remains un-switched over, thus, the finder optical system FS still has the magnification suitable for the standard photography condition.

While moving from the position "c" to a position "d", the gear portion 40a of the bevel gear 40 comes into mesh with the bevel gear portion 39b of the switchover lever 39 thereby pivoting this lever 39 counterclockwise in Fig. 27C. As shown in Fig. 28B, when the gear portion 40a of the bevel gear 40 has reached the position "d" and comes out of the mesh with the bevel gear portion 39b, the switchover lever 39 is backwardly inclined as indicated by a solid line in Fig. 27B.

In this condition, the upper pin 35a of the objective lens holder 35 bound by the forked portion 39a of the switchover lever 39 is positioned rearwardly than the pin 5b of the fixing mount plate 5. Thereafter, as described hereinbefore, by the urging force of the spring 38, the objective lens holder 35 is urged rearwardly and in accordance therewith the switchover lever 39 is further backwardly inclined as shown in Fig. 27C.

With this backward movement of the objective lens holder 35, the objective lens 25 is moved to its rear end of its movable range and at the same time the variable magnification lens 26 is retracted

from the view finder optical path, whereby the magnification of the view finder optical system FS is switched over to be suitable for the tele photography condition. Also, in this condition, the bevel gear portion 39b of the switchover lever 39 has been moved to a position shown in Fig. 28C.

After coming out of the meshing with the bevel gear portion 39b of the switchover lever 39, with the further rotation of the bevel gear 40, the gear portion 40a of the bevel gear 40 reaches a position denoted by a reference mark "e" shown in Figs. 28B and 28C and the rotation of the bevel gear 40 is stopped at this position. In the course of the above operations, the gear portion 40a of the bevel gear 40 does not again come into mesh with the bevel gear portion 39b of the switchover lever 39. Therefore, the magnification of the view finder optical system FS is not changed over and the system FS still has the same magnification suitable for the tele photography condition.

On the other hand, in the photographic optical system PS, while the gear portion 40a of the bevel gear 40 moves from the position "b" shown in Fig. 28A to the position "e" shown in Fig. 28C, the focal length is switched over. And, when the gear portion 40a of the bevel gear 40 is at the position "e", there is provided the initial condition of the standard photography condition. In this condition, the photographic optical system PS is focused at infinity position.

For switching over the focal length from the tele photography condition to the standard photography condition, the bevel gear 40 is rotated counterclockwise from the position "e" in Fig. 28C and its gear portion 40a returns to a position "d" to mesh with the bevel gear portion 39b of the switchover lever 39.

Then, while the gear portion 40a of the bevel gear 40 moves from the position "d" in Fig. 28C to a position "c" in Fig. 28B, the switchover lever 39 is pivoted clockwise in Fig. 27B. As the bevel gear 40 is further rotated and its gear portion 40a comes out of the meshing with the bevel gear portion 39b of the switchover lever 39, the switchover lever 39 is forwardly inclined as indicated by a long and two short dashes line in Fig. 27B. Thereafter, the switchover lever 39 is further pivoted as the objective lens holder 35 is moved forwardly by the urging force of the spring 38 and returns to its forwardly inclined position in the initial condition shown in Fig. 27A.

With this forward movement of the objective lens holder 35, the objective lens 25 is moved to its forward end in its movable range and at the same time the variable magnification lens 26 is inserted into the view finder optical path, whereby the magnification of the view finder optical system FS returns to the magnification suitable for the

divided, there is provided only the pin 18b implanted on the lower face of the delivery lever 18. The rest of the construction are the same as the previous embodiment. Therefore, the construction is illustrated with the same reference marks and will not be particularly described.

Also, in case the focal length of the photographic optical system PS may be switched over in two steps as described in the previous embodiment, the peripheral face of the delivery cam 14 may be divided into more than three portions. Further, it is also possible to arrange such that the focal length of the photographic optical system PS may be switched over in more than three steps. In this case, the peripheral face of the delivery cam 14 may be divided into three or two or four portions or may not be divided at all as well.

Also, in place of the rotary type delivery cam 14 used in the previous embodiment, a translation type may be employed.

Further, in the previous embodiment, the shutter block 4 and the movable lens barrel accommodating the photographic optical system PS are urged forwardly by means of the spring 6 and the fourth pin 18d of the delivery lever 18 shifted in operative connection with the rotation of the delivery cam 14 is engaged into the elongate slot 8a of the movable lens barrel 8 thereby determining the position of the photographic optical system PS. In place of this construction, as shown in Fig. 32, it is also possible to arrange such that the cam follower 18 may move along a cam groove 14b having as its side wall the cam face 14b of the delivery cam 14 coupled with the motor M.

In this Fig. 32, it is to be noted, the peripheral face of the delivery cam 14 is not divided. However, the same construction may be employed even if the peripheral face is divided.

In the case of the sensor used in the previous embodiment, the metal contacts attached respectively on the bottom face of the delivery cam 14 and on the upper face of the cam mount plate 23 come into contact with each other. In place of this, an optical type sensor may be employed, in which a light source and a light receiving element are provided on the upper face of the cam mount plate 23 and on the lower face of the lower half portion 5A of the fixing mount plate 5 and the delivery cam 14 peripherally defines a plurality of holes whereby a position of the cam is detected by using pulse signals generated from the light receiving element with the rotation of the delivery cam 14.

In place of the cam mount plate 23 acting as a fixing member in the previous embodiment, a shaft or the like for supporting the delivery cam 14 may be employed as well.

Furthermore, in the previous embodiment, in the control mechanism for sliding the main lens

unit 4a, which is the fore-element of the photographic optical system PS, towards its recessed position side against the urging force of the spring 6.

As described at page 16 of the present specification, Figs. 33 and 34 show alternative embodiments of the mechanism according to the present invention.

In the previous embodiment, the control mechanism slides the main lens unit 4a, which is the fore-element of the photographic optical system PS, towards its recessed portion against the urging force of the spring 6. A delivery cam 14, which is operatively connected to the motor M, comprises an operating member while the delivery lever 18, which is operatively connected to the shutter block 4 holding the main lens unit 4a, comprises the operated member. There is provided between the delivery cam 14 and the delivery lever 18 a space to permit the movement of the operated member towards its recessed portion. In place of this, the specific construction of the control mechanism may be conveniently varied therefrom as will be illustrated by way of a few examples to be described next.

In a construction shown in Fig. 33, a base plate 45, which holds the fore-element (not shown in Fig. 33) of the photographic optical system PS, is slidable in the forward and rearward directions along a pair of guide rods 46a and 46b. The base plate 45 is urged towards the forward direction by the spring 6. A regulating member 47, which forwardly contacts the base plate 45, regulates the movement of the base plate 45 against the urging force of the spring 6 and its position is adjusted with a male screw element 48.

In this embodiment, the regulating member 47 acts as an operating member while the base plate 45 acts as an operated member. The arrangement for permitting the movement of the operating member (i.e., base plate 45) towards its recessed position is provided by the regulating member 47. The regulating member 47 forwardly comes into a one-sided contact with the base plate 45 to permit a rearward movement of the base plate 45.

Fig. 34 shows still another construction modified from that of Fig. 33. In this construction, the regulating member 47 and the motor M are operatively connected with each other not through a screw member, but through a rack 47a formed on the regulating member 47 and a pinion 49 connected with the motor M. The rest of the construction is the similar in construction as the embodiment of Fig. 33 and therefore like parts, which are denoted by like reference marks, are not further described with regard to their details.

FIG. 1

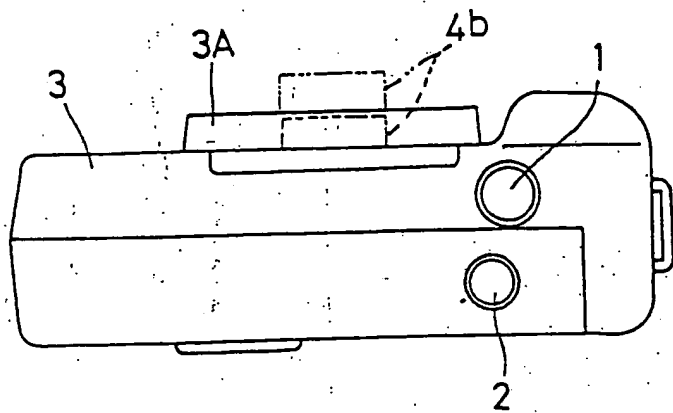


FIG. 30

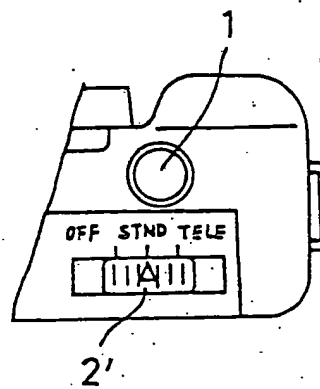


FIG. 3A

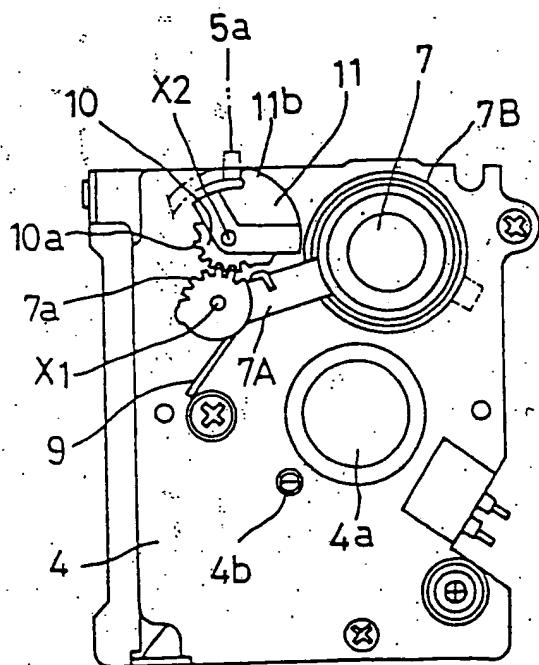


FIG. 3B

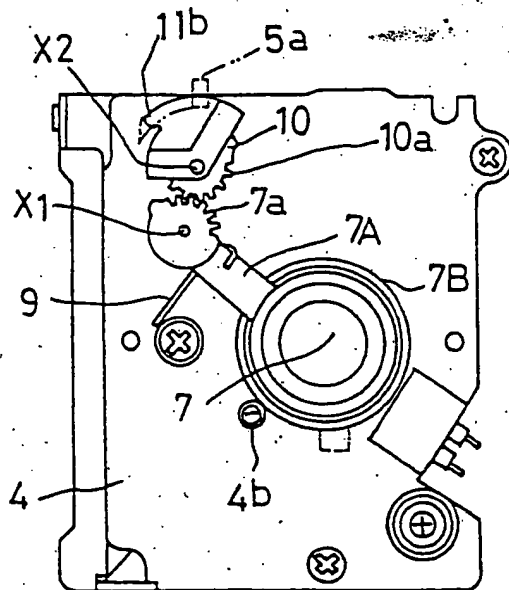


FIG. 2B

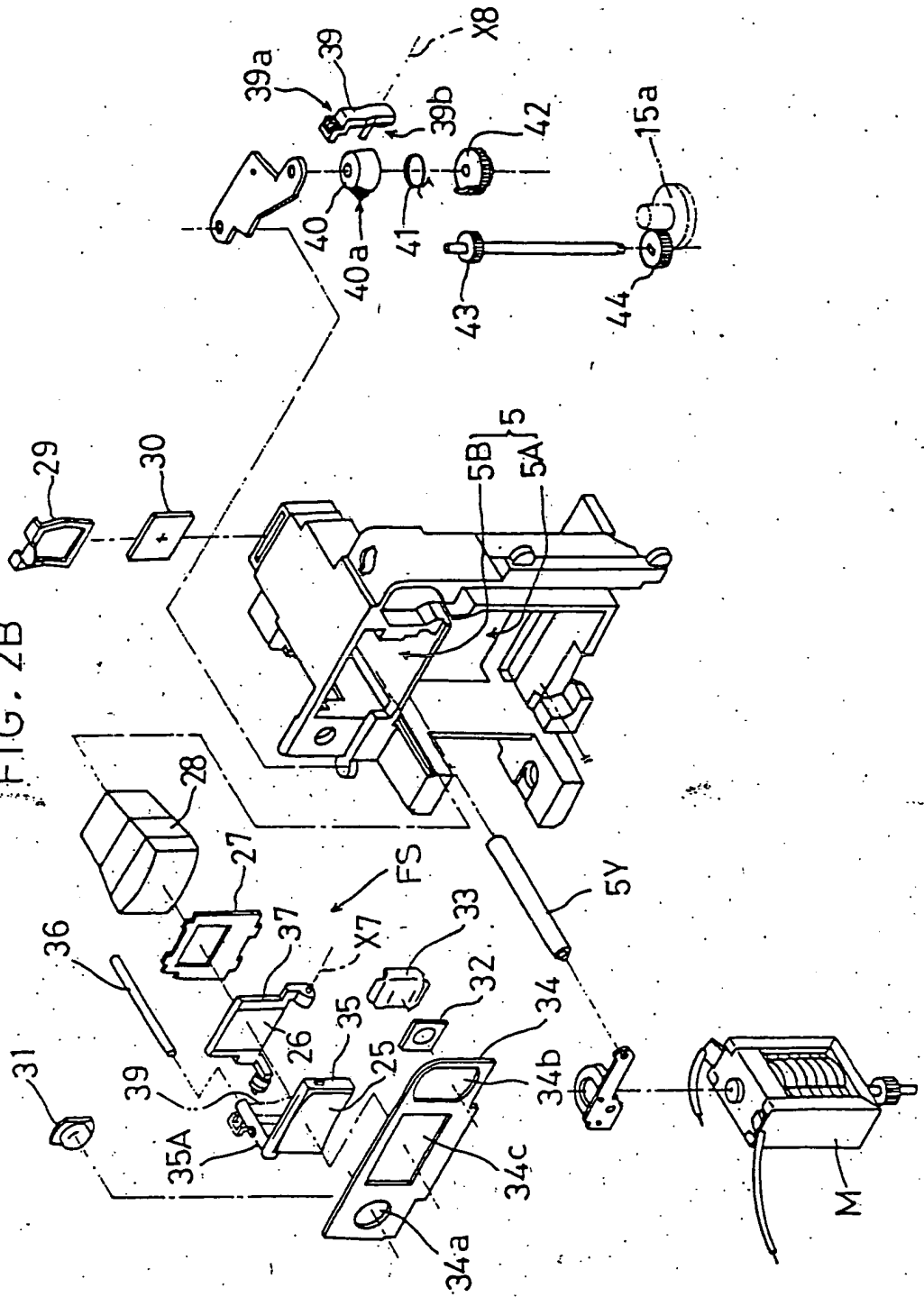


FIG. 6

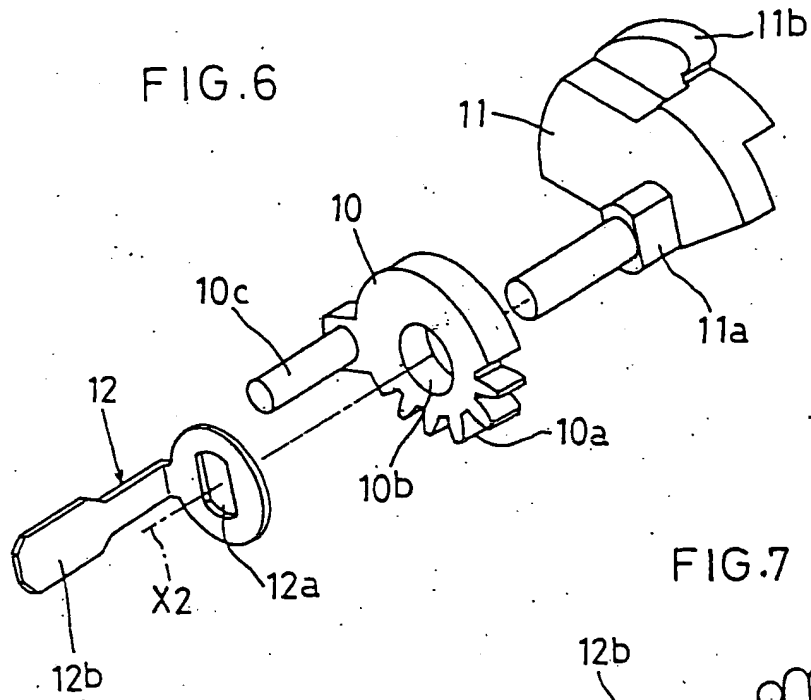


FIG. 7

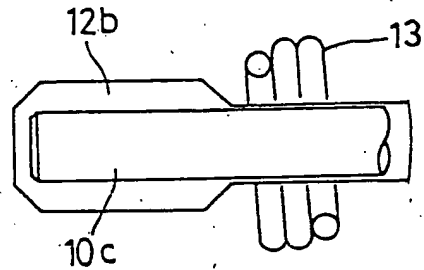


FIG. 8

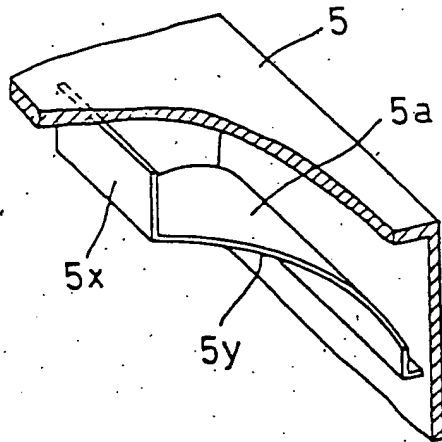


FIG. 10

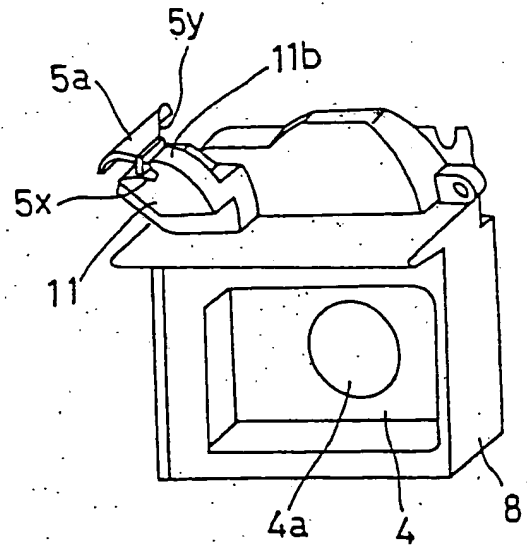


FIG.11

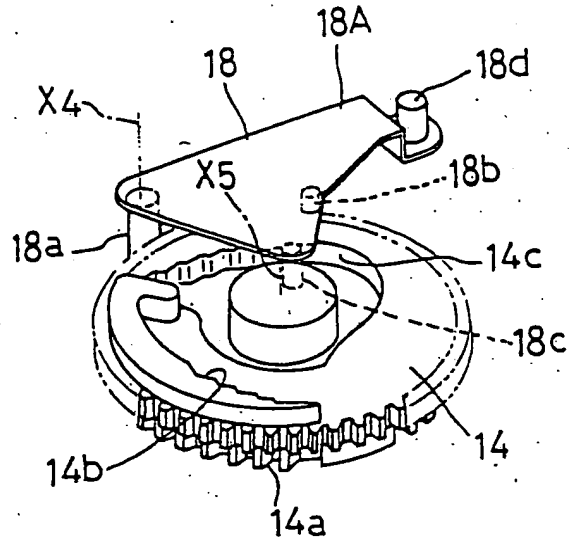


FIG. 12

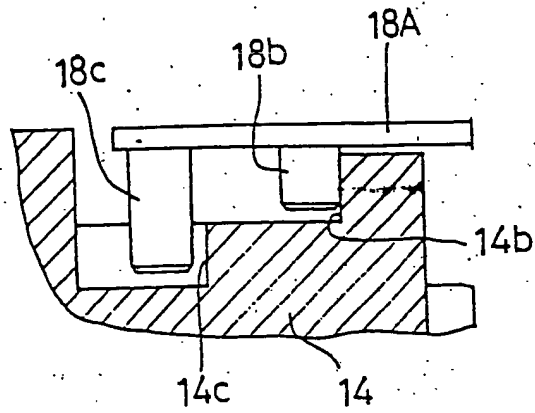


FIG.14

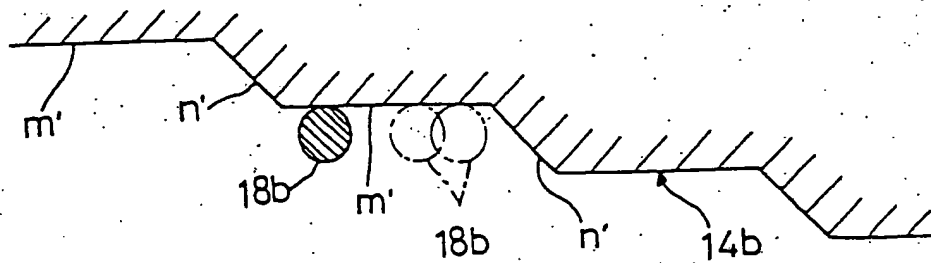


FIG. 15

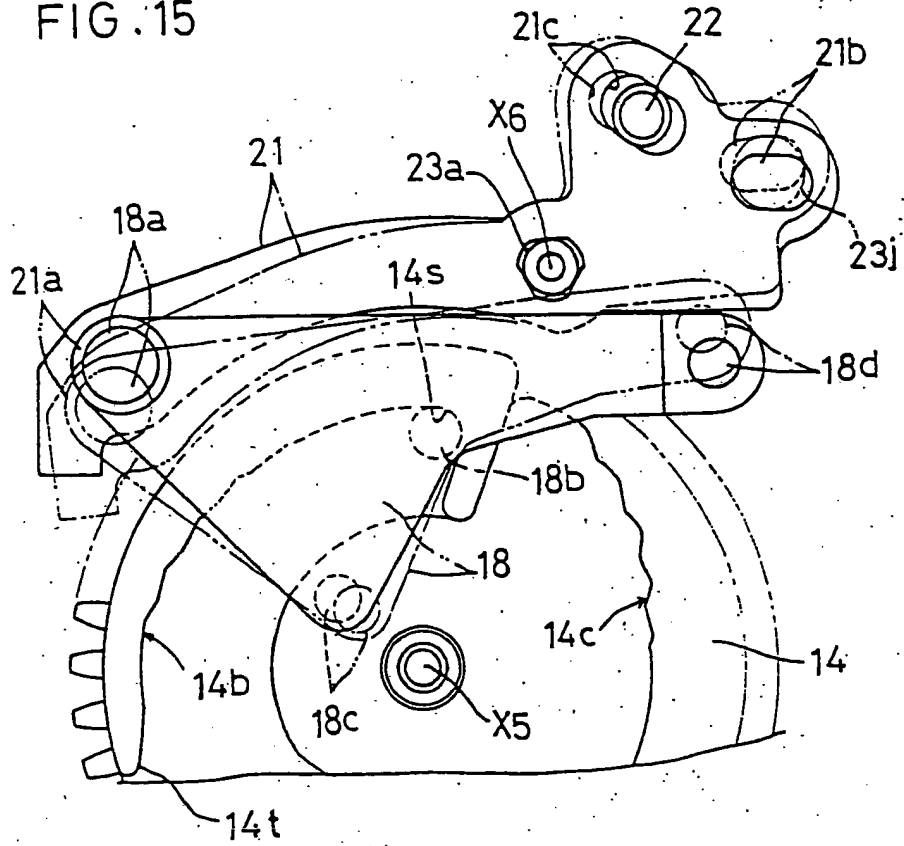


FIG. 16

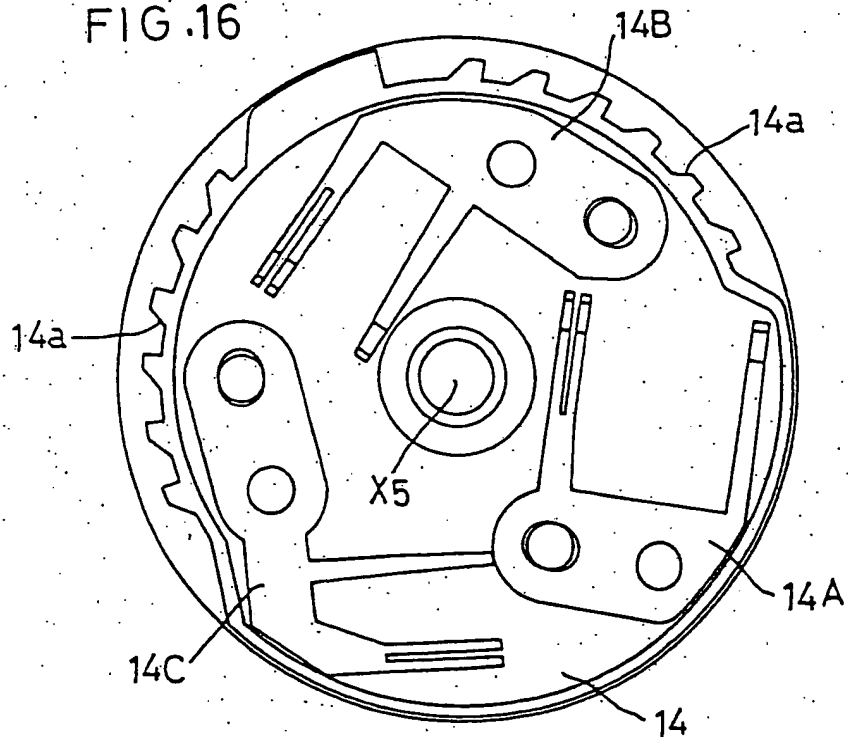


FIG. 19

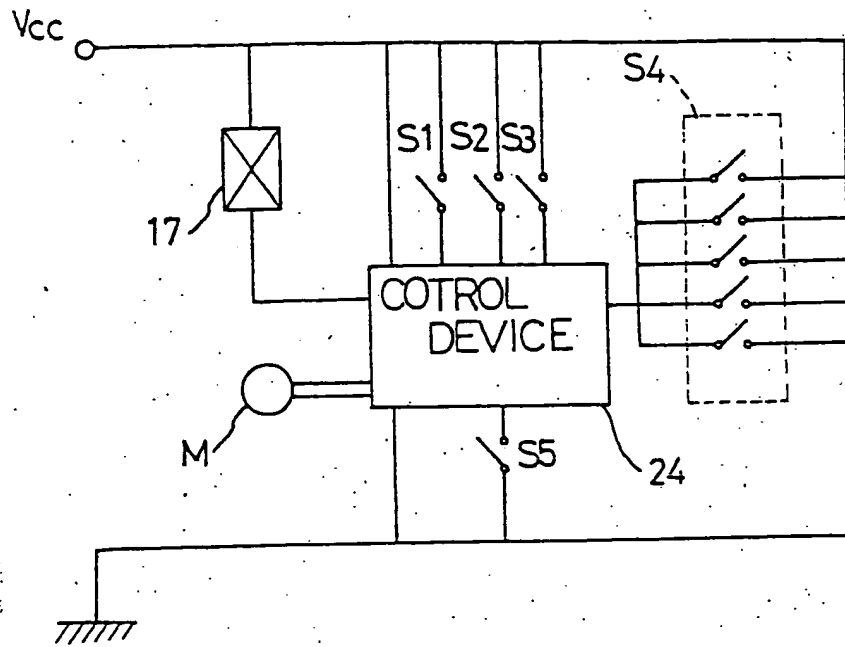


FIG. 20A

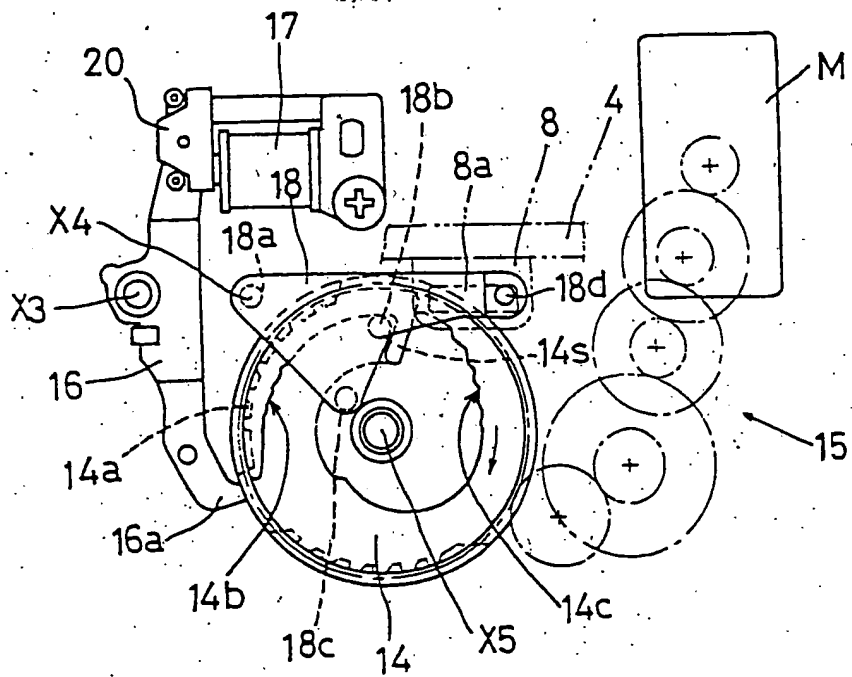


FIG. 20D

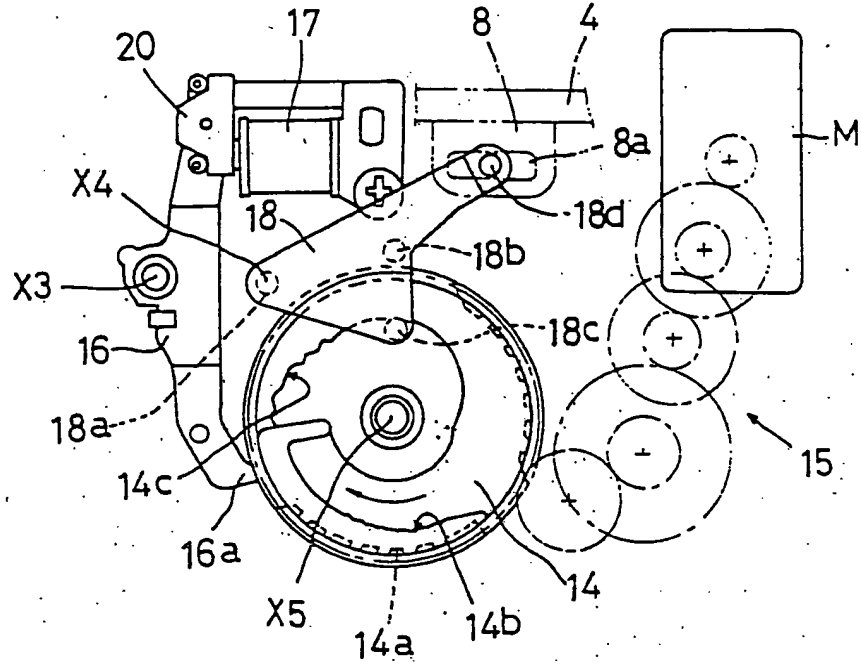


FIG. 20E

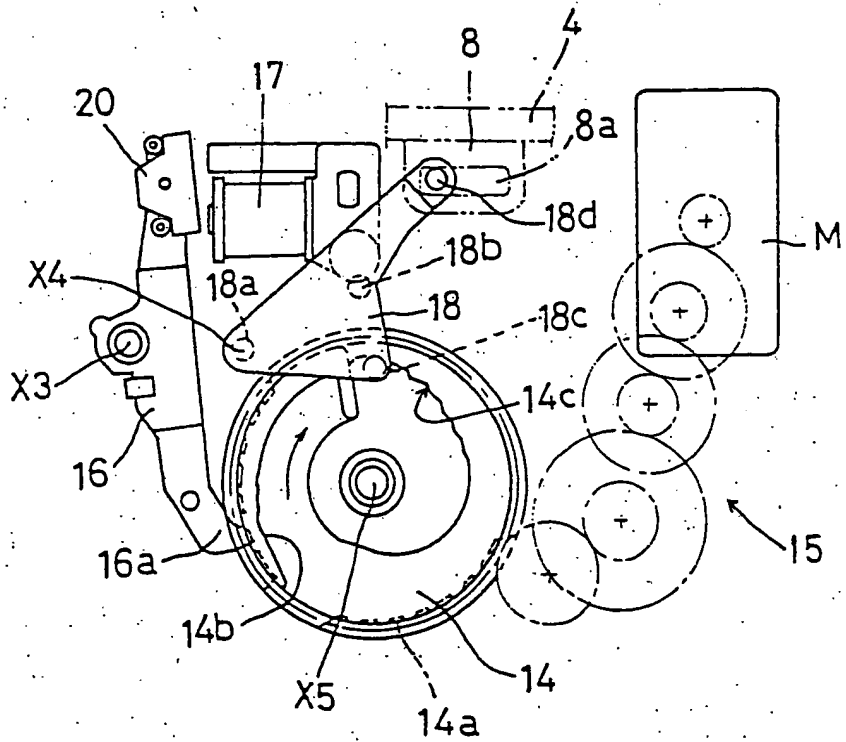


FIG.22

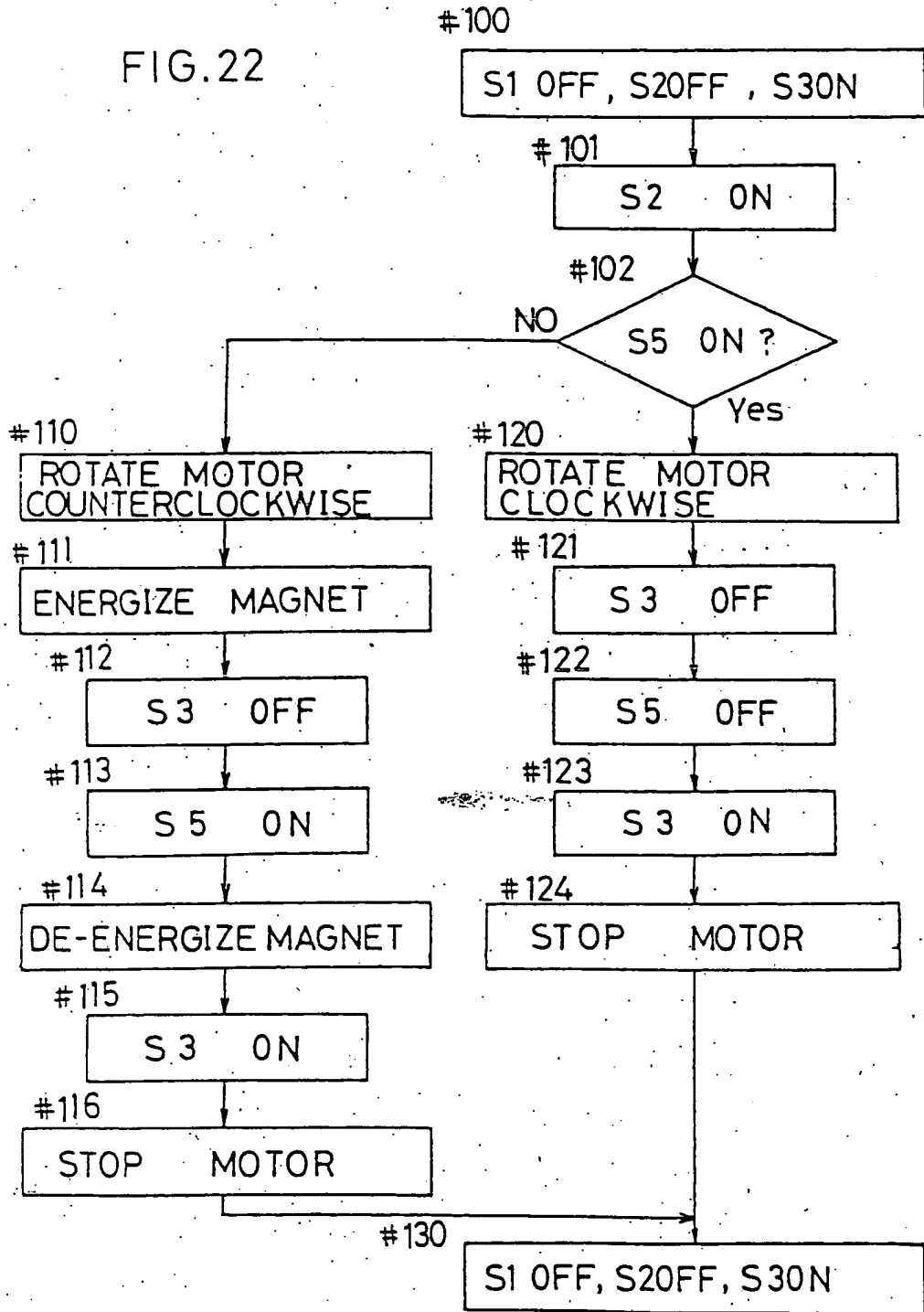


FIG. 24A

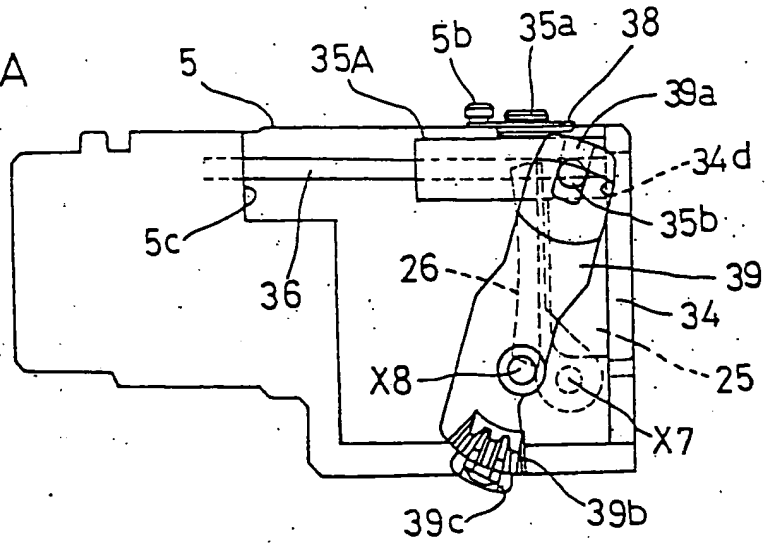


FIG. 24B

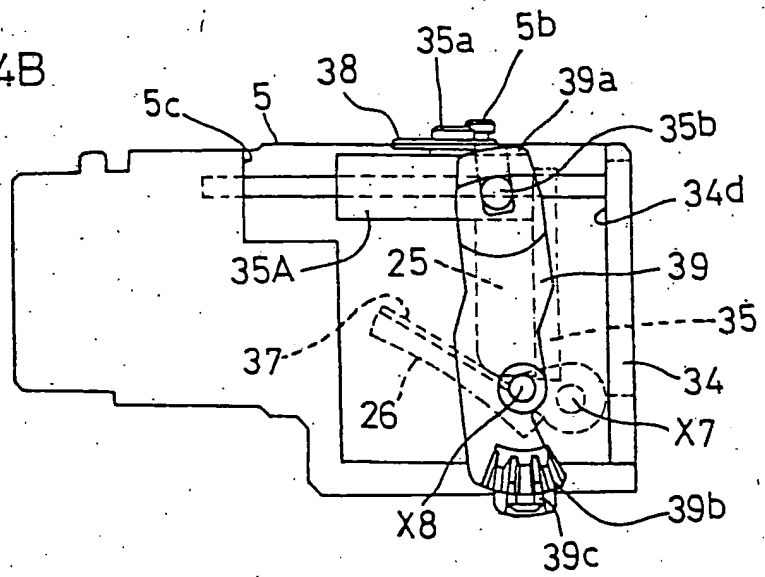


FIG. 24C

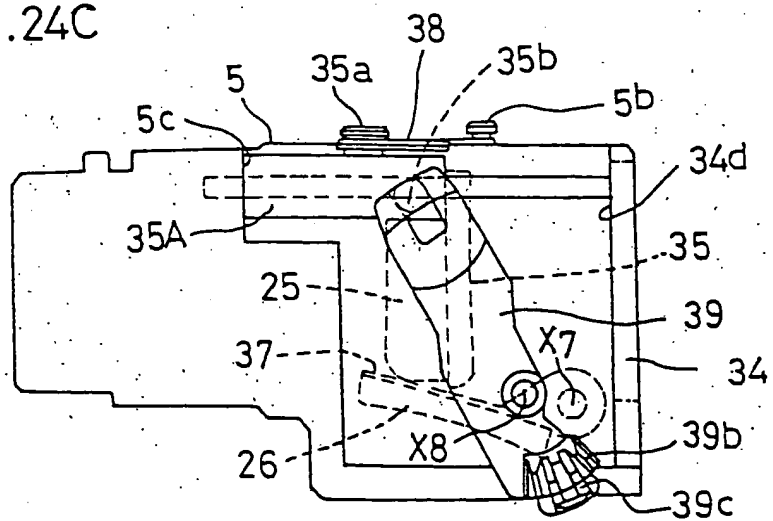


FIG. 26

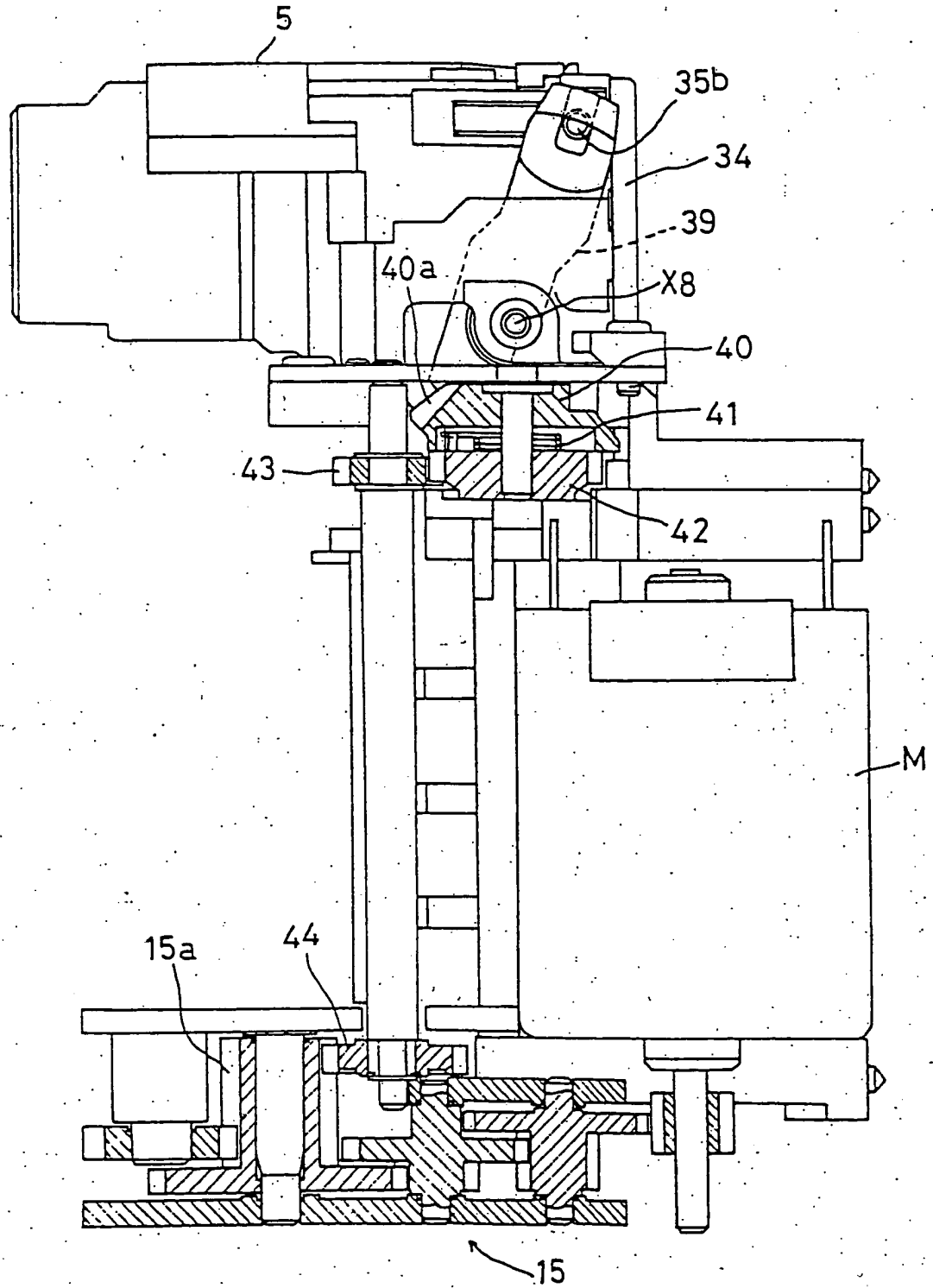


FIG. 31

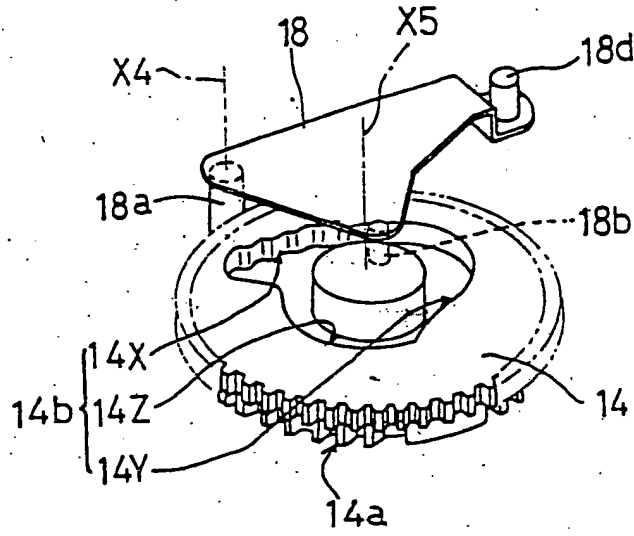


FIG. 32

